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**FLUID MECHANICS OF
COMPRESSION SYSTEM FLOW
CONTROL**



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FLUID MECHANICS OF COMPRESSION SYSTEM FLOW CONTROL

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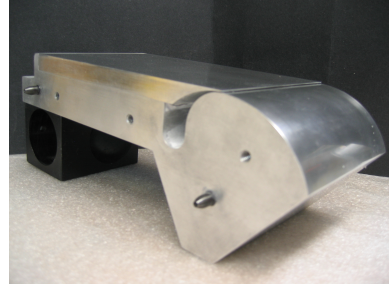
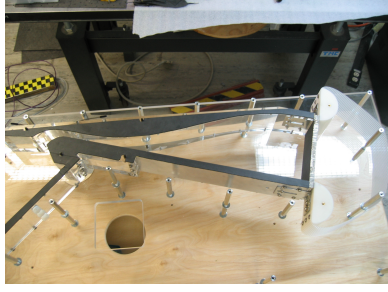
Abstract

The aim of *Fluid Mechanics of Compression System Flow Control* is to increase the diffusion capacity of an axial compressor stator through the application of blade-surface-mounted flow control. The experimental portion of the work employs a small wind tunnel that allows the investigation of flow control concepts applied to a simulated single stator passage. Baseline results are presented along with those of three flow control modules; all are variations of blowing (no aspiration) flow control using a planar jet behind a backward facing step positioned upstream of the passage curvature. Two preliminary investigations are presented. The first includes the effect of the thickness of the lip separating the main stream from the core stream of the backward facing step and its impact on the flow control effectiveness. The second introduces streamwise vorticity via flat plate vortex generators as a means of enhancing the interaction between the blowing jet and the core stream.

Introduction

Flow control particular to axial compression systems has become a very active research topic for the last few decades.¹⁻⁴ It has the potential to open the design envelope for axial compressors to higher loading levels. This translates into higher overall pressure ratios for reduced thrust specific fuel consumption (TSFC). Increased loading levels also increases thrust-to-weight(T/W) ratios by reducing turbine engine axial length for a given pressure ratio. The Air Force Versatile Affordable Advanced Turbine Engine (VAATE) program cycle requirements will drive a much higher overall pressure ratio, thus requiring larger pressure rise per stage in order to keep overall engine length reasonable.

The research objective of this work is to increase the diffusion capability of a stator flow passage via flow control methods. To assist in this effort, an experimental flow control research rig has been designed and will be briefly outlined in this report. Following will be the initial investigation and comparison of three flow control modules; all are variations on a theme of blowing only flow control using a planar jet behind a backward facing step. Two of the variations are preliminary investigations into the effect of the separation distance between the primary and secondary streams due to finite material thickness separating the streams. The third variation seeks to embed streamwise vorticity via vortex generators into the shear layer between the primary and secondary flow streams. The goal is to enhance the interaction between



(a) Top View of FCAD wind tunnel (b) Flow control module with slot

Figure 1: FCAD Wind Tunnel

the streams and thereby reduce the secondary flow requirements for a given level of diffusion.

Experimental Apparatus

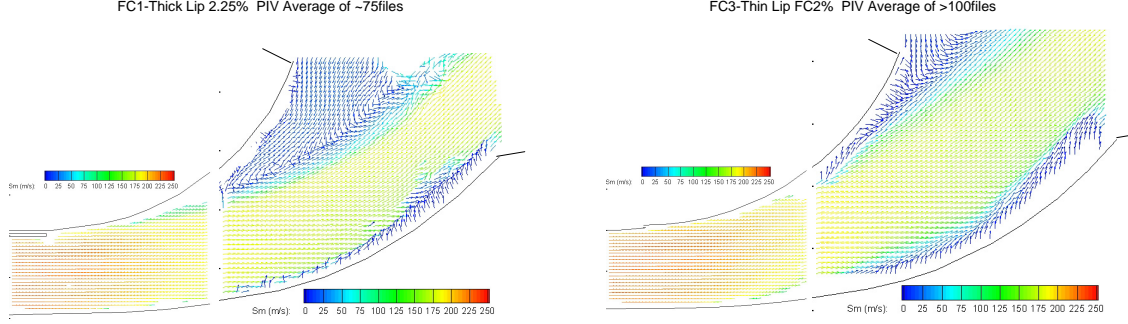
The Flow Control Augmented Diffusion (FCAD) wind tunnel is an atmospheric in-draft design. After passing through a bellmouth and flow straightener, the airflow accelerates through a converging inlet of roughly 12:1 area contraction to a rectangular throat, 1.5 x 10.2 cm (0.6 x 4.0 in). The throat, typically operated at Mach 0.7, is the beginning of the curved diffuser test section. The diffuser geometry is based on aggressive goals for an axial compression system. The diffuser passage has an exit-to-throat area ratio of 2.92, a flow turning angle of 70 degrees, with suction (convex) side radius of curvature nearly constant at 5.1 cm (2.0 in). Following the diffuser is a sudden expansion into a rectangular settling chamber. An adaptor piece guides the flow from the settling chamber into a flexible 7.6 cm (3 in) diameter duct connected to the primary flow driver. Figure 1 illustrates the basic flowpath from the inlet (at right) to the dump chamber (left).

Because present research goals include identification of key physical mechanisms relating diffusion and flow control, the tunnel was designed with optical access as a priority. The top and bottom walls are transparent acrylic sheets, and sandwiched in between are wall segments with height of 10.2 cm (4 in). Defining the suction (convex) side of the curved diffuser test section is a readily replaceable aluminum module for testing various flow control concepts. One such module is shown in Figure 1. The planar jet height for these studies was fixed at 0.508 mm and the jet surface was constructed as follows: 1) The baseline convex surface was a constant radius of 50.8 mm ; 2) This radius was reduced 1.397 mm at the throat to accommodate the jet and lip thickness; 3) A linear function in the angular coordinate was used to fit this radius to the 50.8 mm radius at the exit of the passage.

Lip Thickness Study

The influence of the separation distance between the core flow and jet flow due to the finite material thickness of the upper plate on the flow field was investigated. Two material thickness were tested: 0.635 mm and 0.127 mm which are 125% and 25% of

the planar jet size respectively. The 0.127 mm thickness was achieved by honing the upper surface of the 0.635 mm plate at a 5 degree angle. Figure 2 shows the post processed DPIV average flowfield for a flow fraction of approximately 2% for both the 125% and 25% cases. The 25% lip thickness shows much more diffusion over the 125% case.



(a) Composite image of 125% primary to secondary separation distance and 2.25% flow control

(b) Composite image of 25% primary to secondary separation distance and 2.00% flow control

Figure 2: Effect of Core and Jet Separation on Diffusion Effectiveness

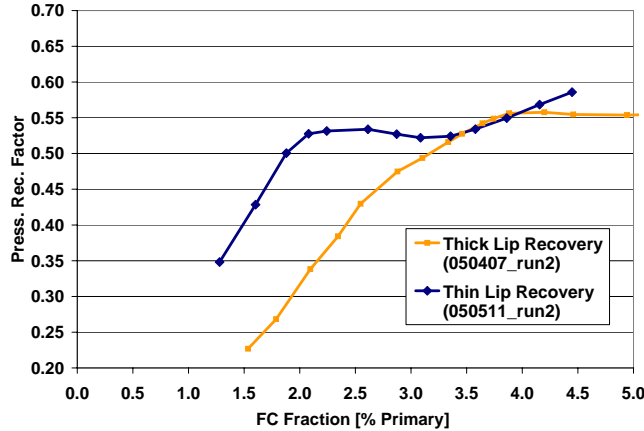


Figure 3: Effect of Core and Jet Separation on Static Pressure Recovery

As is clear from Figure 2, larger regions of flow separation exist for the 125% thickness case. Consequently, the 25% lip thickness requires half the secondary flow requirement for the same static pressure rise coefficient versus the 125% case as seen in Figure 3. This has significant implications regarding the manufacturing requirements for such an approach.

Streamwise Vorticity Addition

It is postulated that increased streamwise vorticity will enhance the mixing between the main and secondary streams resulting in a persistent momentum exchange that will significantly delay separation. In order to effect this condition, pairs of counter-rotating streamwise vortices are introduced via flat plate vortex generators placed on the upper surface of the flow control module plate as shown in Figure 4. Vortex pairs were created such that their induced motion was toward the surface and the planar jet.^{5,6}

The desired circulation strength, Γ , and spacing, d , for the vortex pair was determined by taking a much simplified approach to the flowfield. The flowfield was approximated as undergoing a transition from velocity V_1 at the throat to V_2 at the exit in a linear fashion. A relationship between streamwise position and time could then be obtained. An approximation of the circulation generated by a NACA 0012 vortex generator⁷ was used to relate the VG geometry to circulation strength. It was also assumed that the streamwise vortex pair advects with the background flow field. Due to space limitations, the final result will only be presented here. A functional relationship in the form of an inequality was generated relating the geometric parameters of the VG with the diffusion characteristics of the passage:

$$\begin{aligned}\frac{F(AR, d/C)G(\alpha, h/\delta)}{H(DF)} &\leq 1 \\ F(AR, d/C) &= \left(\frac{AR + k_2}{k_1}\right) \left(\frac{d}{C}\right) \\ G(\alpha, h/\delta) &= \left(\frac{1}{\alpha}\right) \left(\frac{1}{\tanh(k_3(h/\delta))}\right) \\ H(DF) &= -\frac{4}{\pi^2} \ln \frac{(1 - DF)}{DF}\end{aligned}$$

where

$$k_1 = 1.61; k_2 = 0.48; k_3 = 1.41; k_4 = 1.00; AR = \frac{8h}{\pi c}; DF = 1 - \frac{V_2}{V_1}$$

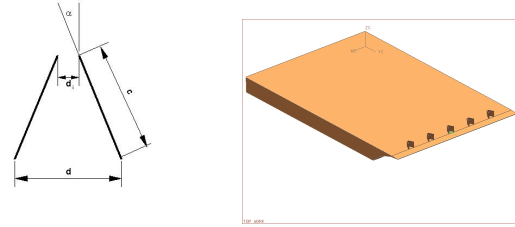
where C is the surface arc length; c is the VG chord length; AR is the VG aspect ratio; h is the VG height; δ is the boundary layer height at the VG leading edge; d is the distance between VG trailing edges (see Figure 4); V_1 is the passage throat velocity; V_2 is the passage exit velocity; and DF is a simplified diffusion factor as defined above.

The function H is dependent only upon flow field kinematics (the diffusion factor). Both functions F and G are dependent on geometric characteristics along with the boundary layer thickness. A family of characteristic curves of FG/H were produced by fixing the function G through a choice of α and h/δ and the diffusion factor of 0.7 for the passage under consideration. Two geometric constraints were applied due to physical limitations of using these VG's. The first was due to the constraint of

generating two counter-rotating vortices a distance d apart using VG's at an angle of attack α . Figure 4 shows a top-down view of the VG's with relevant dimensions. The second geometric constraint was derived from a heuristic argument that the distance, d , between the vortex pair should be less than the height of the VG. This is applied so that the vortex image does not dominate initially and cause the vortex pair to migrate apart. A geometric configuration was chosen that maximizes the circulation and yet is acceptable according to the two geometric constraints. The boundary layer thickness for the wind tunnel at the throat was approximated from a CFD calculation at 0.254 mm. Using the surface arc length of the passage, C , of 31 mm gives the following VG geometric parameters:

$$h = 3.81mm; d = 3.765mm; c = 3.962mm; \alpha = 25^\circ, d_i = 0.3765mm$$

Figure 4 shows the flow control module upper plate obtained by using these geometric parameters at a spacing of $4.5d$ between VG pairs. The spacing between pairs of vortices was chosen based on guidelines outlined in 5.



Future Direction

The embedded streamwise vorticity experiment will be tested and stereo PIV measurements take in order to analyze the streamwise vortex generation and migration. Also, exit total pressure measurement traverses will be added to the experimental data collection. Methods to address endwall effects will also be considered and modeling performed to enhance the data obtained from experimental measurements.

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Publications

Gorrell, S. E., Car, D., Puterbaugh, S. L., Estevadeordal, J., and Okiishi, T. H., “An Investigation of Wake-Shock Interactions in a Transonic Compressor with DPIV and Time-Accurate CFD,” ASME Paper GT2005-69107, June, 2005.

McCray, T.W., Estevadeordal, J., and Puterbaugh, S.L., “Parallel Computing for Linux Clusters: Application to Particle Image Velocimetry,” 43rd AIAA Aerospace Sciences Meeting, Reno, NV, January, 2005.

Estevadeordal, J., Goss, L., Car, D., and Bailie, S.T., “PIV with LED: Particle Shadow Velocimetry,” 43rd AIAA Aerospace Sciences Meeting, Reno, NV, January, 2005.

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